AMENDMENTS TO THE SPECIFICATION:

Page 15:

Please substitute the following paragraph for the paragraph beginning at line 17:

Also, in a fifth aspect of the pulley support double row ball bearing according to the present invention, a back-to-back duplex type contact angle is given to each of the balls arranged in a double row, and an inner diameter of the outer ring on the axially outside portion, being an anti-loading side, of each of the outer ring raceways is greater than or equal to the largest diameter of each of the outer ring raceways.

Page 23:

Please substitute the following paragraph for the paragraph beginning at line 13:

Here, while omitted from the figure, a concave part which concaves radially inwards, is formed on one part of the outer circumferential surface of the rim 48 of each of the retainers 45, and by accumulating the grease in this concave part it is also possible to ensure the amount of grease disposed within inner space 47. Moreover, a concave part which concaves radially outwards, is formed on one part

of the connecting portion 52 existing on the portion near both ends of the inner circumferential surface of the outer ring 40 and by accumulating the grease in this concave part it is also possible to ensure the amount of grease disposed within inner space 47. In either case, on the part corresponding to the concave part, it is desirable to set the spacing in the radial direction between the outer circumferential surface of the retainer and the inner circumferential surface of the outer ring to more than or equal to 15% of the diameter of the balls 44, from the perspective of ensuring the amount of grease.

Page 31:

Please substitute the following paragraph for the paragraph beginning at line 12:

Next, FIG. 6 illustrates a fourth example of an embodiment of the present invention, corresponding to a sixth aspect. In the case of a double row ball bearing 32d of the present example, a face-to-face duplex type contact angle is given to each of the balls 44 arranged in the double row. To match this, on the inner circumferential surface of the outer ring 40b, there is formed a pair of outer ring raceways 41b, being angular type with each facing inwards in the axial direction.

Furthermore, the inner diameter of the outer ring 40b at the interval portion between the outer ring raceways 41b, being the anti-loading side, is greater than or equal to the largest diameter of each of the outer ring raceways 41b. That is to say, the inner diameter of the outer ring 40b is the smallest at both outside ends of both of the outer ring raceways 41b, and at the interval portion between both of the outer ring raceways 41b, the inner diameter is set larger than both end portions such that a so-called groove depth is zero.